

Actuator Re-design Example

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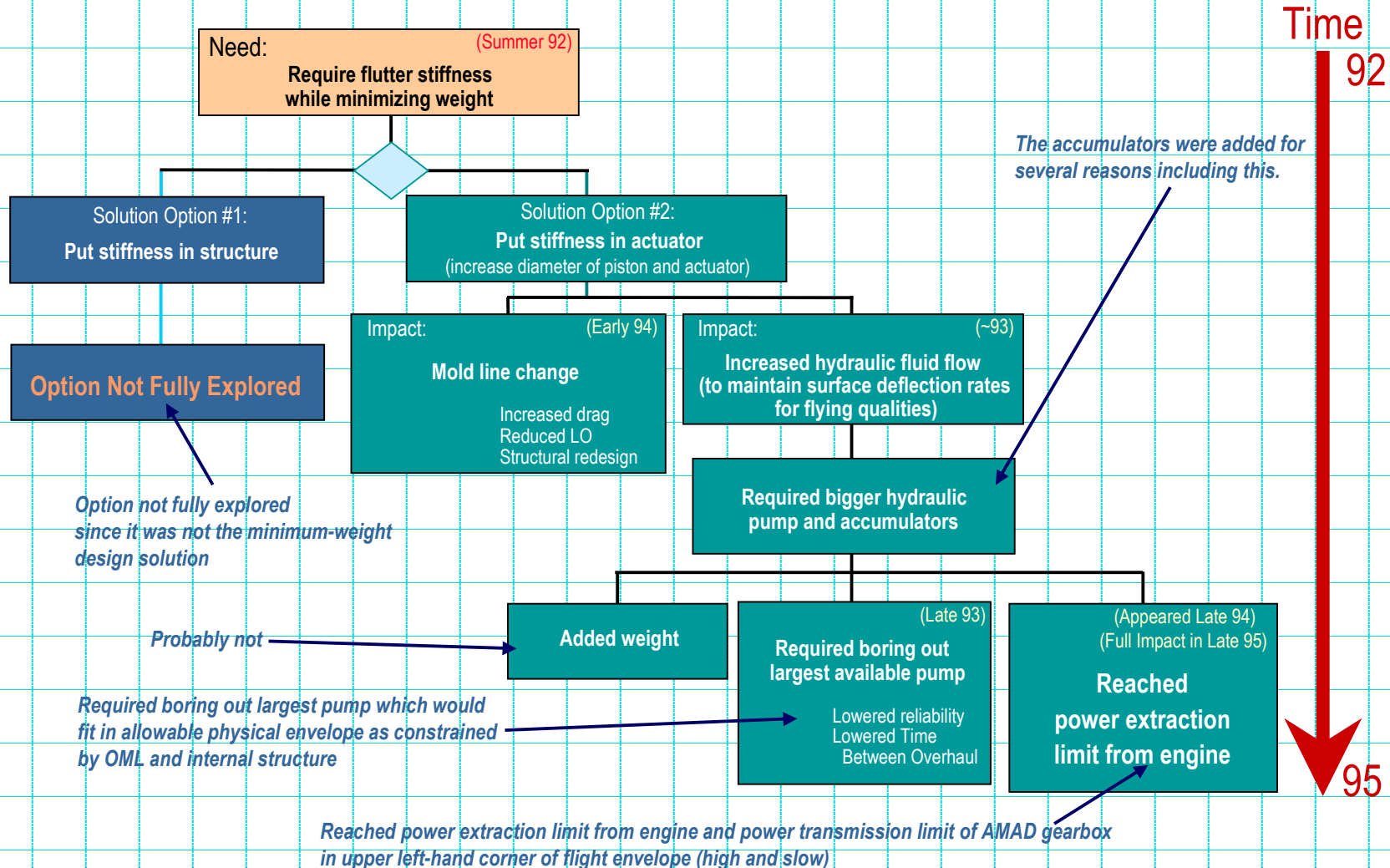
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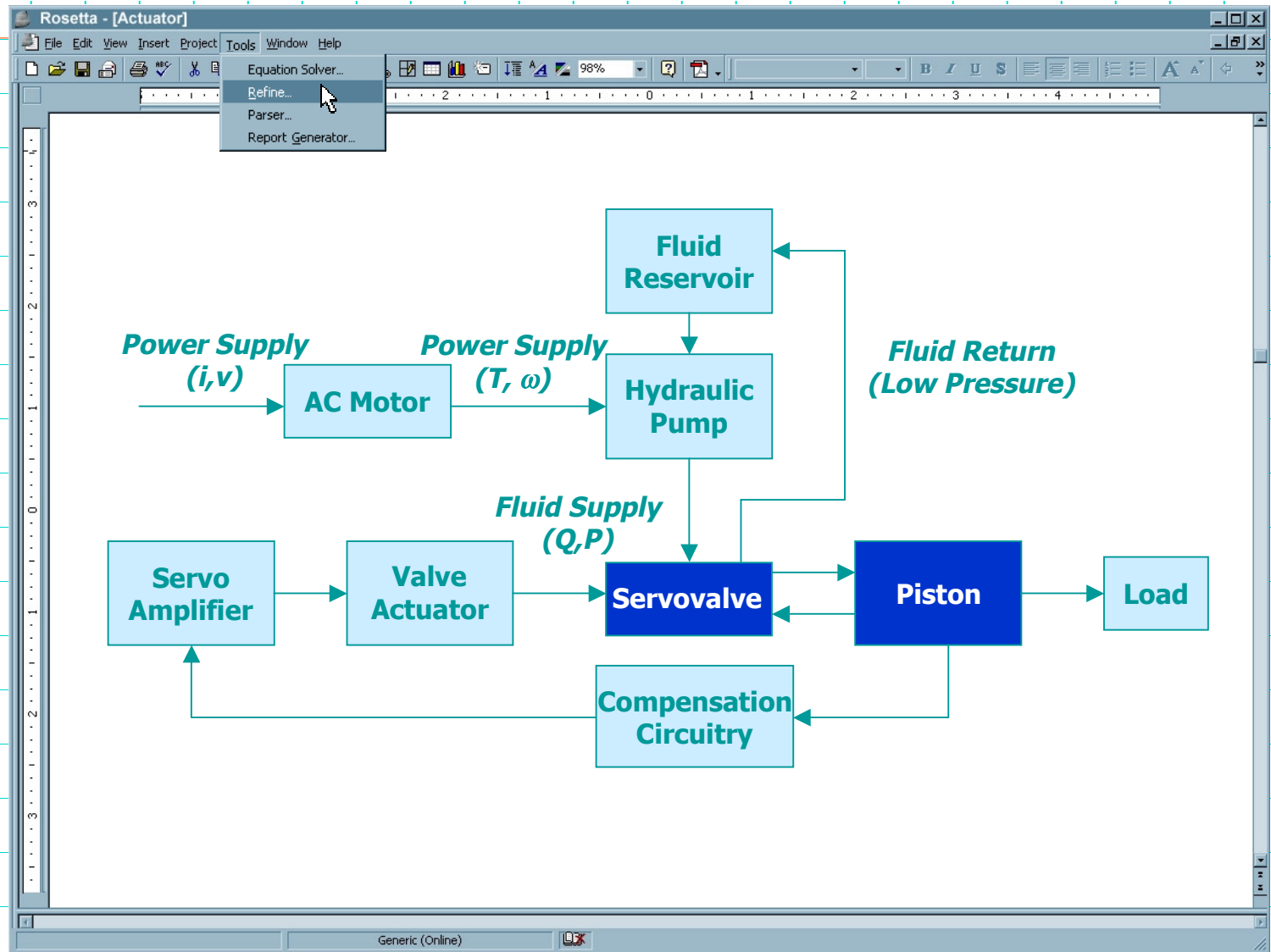
Actuator Redesign

- **Goal: Simulated re-creation of real-life actuator redesign problem**
- **Achievements:**
 - Developed Rosetta design model of servovalve, cylinder and actuator
 - Developed power constraint and functional models
 - Hand translated Rosetta models into MATLAB system representations
 - Used interactions to represent constraint and functional model interaction
 - Used MATLAB model to demonstrate early detection of constraint violation
- **Status:**
 - Actuator problem analyzed and Rosetta models written
 - Generated interaction result between power and functional models
 - Analytically predicted power constraint violation based on MATLAB models

Prior Airplane Design Experience with Altering an Existing Design



Using Systems Design Tools – Actuator



Design Assumptions / Limitations

- Specifics of the operating modes are not available
- Only the servovalve and cylinder piston are modeled
- Power Assumptions
 - Maximum engine power approx 2.4×10^7 ft-lbs/s
 - 0.5% or less of engine power available to power hydraulic system
 - About 12.5% of hydraulic power used in an actuator
 - Maximum power available to actuator is about 15,000-20,000 ft-lbs/sec
- Efficiency Assumptions
 - Engine--90%
 - Hydraulic pump--60%
 - Servovalve--90%
 - Hydraulic cylinder—99%
- Modeling done in conjunction with KU Aerospace department faculty

Functional Servovalve Equations

$$\frac{Q}{Q_{\max}} = \frac{U}{U_{\max}} \sqrt{1 - \frac{P}{P_s} \operatorname{sgn}\left(\frac{U}{U_{\max}}\right)},$$

where

$$\frac{Q}{Q_{\max}} = \% \text{ max flow rate achieved}$$

$$\frac{U}{U_{\max}} = \% \text{ max valve opening achieved}$$

P = differential pressure

P_s = source pressure

Functional Servovalve Equations (cont)

$$\text{power} = P_s Q_s - P_2 Q_2 + P_1 Q_1$$

$$\left. \begin{array}{l} \left| \frac{Q}{Q_{\max}} \right| \leq 1 \\ \left| \frac{U}{U_{\max}} \right| \leq 1 \\ \left| \frac{P}{P_s} \right| \leq 1 \end{array} \right\} \text{constraints}$$

Rosetta Functional Servovalve Equations

```
begin continuous
```

```
//Nonlinear steady state valve equation.
```

```
F1: Q/Q_max = U/U_max *  
      ((1 - P/P_S) *  
      sgn(U/U_max))^0.5;
```

```
//Flow, spool disp, diff pressure cannot exceed  
//max
```

```
C1: abs(Q) =< abs(Q_max);
```

```
C2: abs(U) =< abs(U_max);
```

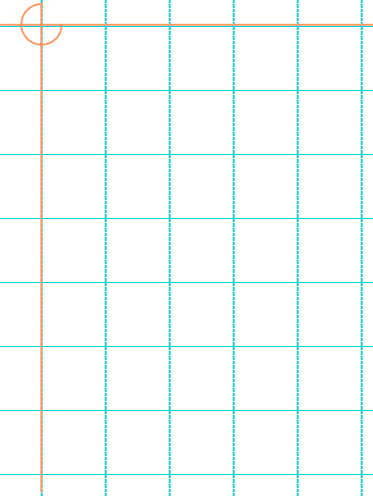
```
C3: abs(P) =< abs(P_S);
```

```
end servovalve_fcn;
```


The Redesign Problem

- **Additional force required to overcome flutter in high speed operating mode**
 - Differential pressure relatively low due to high speed operation
- **Power obtained by increasing piston area in the cylinder**
 - Assumed starting point of 2.5in^2
 - Increased to 2.75in^2
- **Actuator power budget assumed to be 15,000-20,000 ft-lbs/sec**
- **Functional model indicates no problems and additional force is obtained**

Results of MATLAB Model Run



Interaction With the Power Model

- The actuator has an associated power limitation of 15,000-20,000 ft-lbs/sec
 - The power constraint model is expressed is separated from